

Theoretical Analysis on the Viscoplastic Buckling of Local Sharp-notched SUS304 Stainless Steel Tubes under Cyclic Bending

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Abstract

In this paper, the theoretical analysis for simulating the buckling of local sharp-notched SUS304 stainless steel tubes with different notch depths subjected to cyclic bending at different curvature rates is presented. Experimental data of tubes with notch depths varying from 0.2 mm to 1.0 mm were considered. Three different curvature rates, 0.0035, 0.035 and 0.35 $m^{-1}s^{-1}$, were used in the experimental test [1]. The experimental controlled curvature-number of bending cycles required to ignite buckling relationships on a log-log scale demonstrated that for a fixed curvature rate, five almost parallel lines corresponding to five different notch depths were observed. In addition, the slopes of the parallel lines for three different curvature rates were almost the same. Finally, the empirical formulation proposed by Kyriakides and Shaw [2] was modified for simulating the aforementioned relationships. It was found that the experimental and analytical data agreed well.

Keywords: Local Sharp-Notched SUS304 Stainless Steel Tubes, Notch Depths, Curvature Rates, Cyclic Bending, Controlled Curvature, Number of Bending Cycles Required to Ignite Buckling.

1. Introduction

In 1998, Pan's research team started, experimentally and theoretically, to explore different kinds of tubes subjected to monotonic bending or cyclic bending given different loading and geometry conditions. For example, Pan and Her [3] experimentally investigated the viscoplastic failure of SUS304 stainless steel tubes subjected to cyclic bending. To simulate the relationship between the cyclic controlled curvature and the number of bending cycles required to yield buckling, an empirical form was introduced by them. Lee et al. [4] studied the influence of the D_o/t ratio on the response and stability of circular tubes subjected to symmetrical cyclic bending. Diverse Diameter-to-thickness ratios of SUS304 stainless steel tubes were investigated and an empirical form was suggested for describing the correlation between the cyclic controlled curvature and number of bending cycles required to generate buckling. Lee et al. [5] inspected the cyclic bending stability of 316L stainless steel

tubes. The endochronic constitutive model combined with the principle of virtual work was employed to predict the moment-curvature and ovalization-curvature relationships. In addition, Chang and Pan [6] also conducted experiments through which circular tubes' degradation and stability for different outer diameters were investigated. They discovered that the curve of the ovalization vs. number of bending cycles could be separated into initial, secondary and tertiary stages.

Tubes are typically used in harsh environments, which may corrode the surfaces and create notches. Notched tubes should exhibit responses and collapse mechanisms different from their smooth-surfaced counterparts. From 2010, Pan's research group started experimental and theoretical investigations on the response of sharp-notched tubes under cyclic bending. Lee et al. [7] experimentally examined the change in ovalization along with the number of bending cycles for sharp-notched circular tubes subjected to cyclic bending. Three stages (initial, secondary, and tertiary) were clearly observed from the curve of ovalization versus the number of bending cycles. Later, Lee [8] investigated the behavior and failure of sharp-notched SUS304 stainless steel tubes subjected to cyclic bending. Asymmetry, ratcheting, and increasing ovalization-curvature curves were discovered. In addition, Lee et al. [9] experimentally examined the viscoplastic buckling of sharp-notched SUS304 stainless steel circular tubes under cyclic bending, and changes in both the notch depth and curvature rate were examined. Observations of a certain curvature rate revealed that the cyclic-controlled curvature and the number of bending cycles required to yield buckling relationships at a log-log scale exhibited parallel lines for every notch depth.

Although the viscoplastic response and collapse of sharp-notched SUS304 stainless steel tubes under cyclic bending has been investigated [9], but the type of sharp notch was the circumferential sharp notch as shown in Fig. 1. However, tubes in actual applications are typically used in cruel environments, which may corrode the tube's surface and create a local sharp notch. The response and collapse of local sharp-notched tubes submitted to cyclic bending at different curvature rates should exhibit differing from that of circumferential sharp-notched tubes submitted to cyclic bending at different curvature rates.

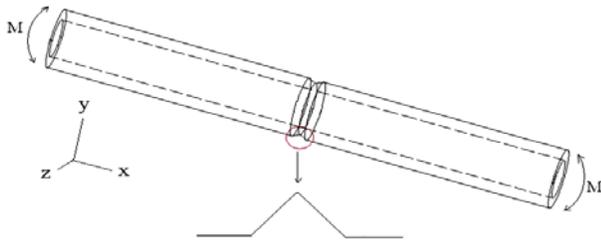


Fig. 1 A schematic drawing of a circumferential sharp notch.

In this paper, the theoretical analysis for simulating the buckling of local sharp-notched SUS304 stainless steel tubes with different notch depths subjected to cyclic bending at different curvature rates is presented. Experimental data tested by Lee et al. [1] were compared with the theoretical analysis. The empirical formulation proposed by Kyriakides and Shaw [2] was modified for simulating the aforementioned relationships. Good agreement between the experimental and theoretical results has been achieved.

2. Experiments

2.1 Experimental Devices

Fig. 2 schematically shows the experiment executed by a specially built tube-bending machine. This facility was set up to conduct monotonic, reverse, and cyclic bending tests. A detailed explanation of the experimental facility can be found in many papers (e.g., Pan and Her [3], Lee et al. [4]). Pan et al. [10] designed a new light-weight apparatus to measure the curvature and ovalization of the tube shown in Fig. 3. Two side-inclinometers in the apparatus were used to detect the angle variation of tubes during cyclic bending. The amount of curvature can be determined by a simple calculation according to angle changes.

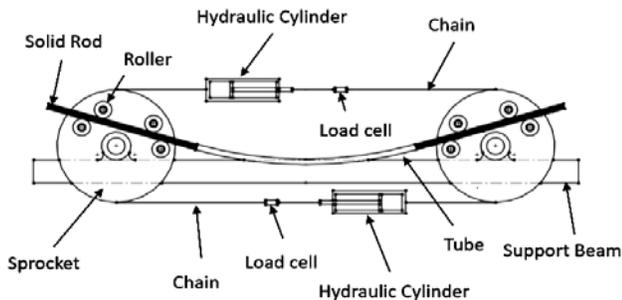


Fig. 2 A schematic drawing of the tube-bending machine.

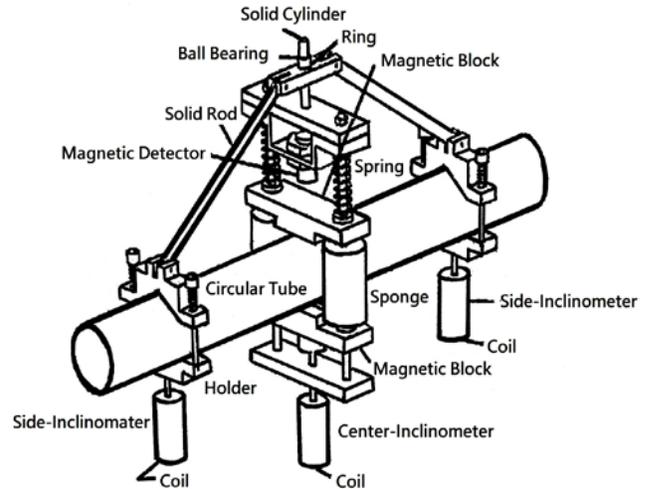


Fig. 3 A schematic drawing of the curvature-ovalization measurement apparatus.

2.2 Material and Specimens

The circular tubes used in this study were made of SUS304 stainless steel. The ultimate stress, 0.2% strain offset the yield stress and the percent elongation are 626 MPa, 296 MPa and 35%, respectively. The raw smooth SUS304 stainless steel tube had an outside diameter D_o of 36.6 mm and wall-thickness t of 1.5 mm. The raw tubes were machined on the outside surface to obtain the desired local notch depth (a) of 0.2, 0.4, 0.6, 0.8 and 1.0 mm. Fig. 4 shows a schematic drawing of the local sharp-notched tube.

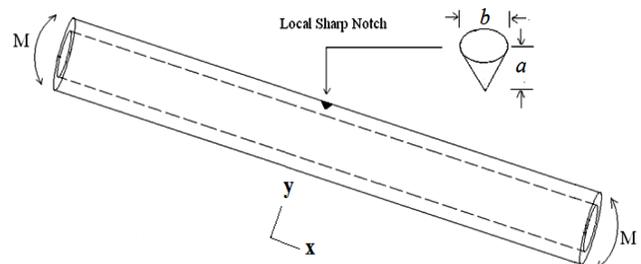


Fig. 4 A schematic drawing of a local sharp-notched tube with a notch depth of a .

2.3 Test Procedures

The bending experiments were performed under curvature-controlled conditions. The curvature rates ($\dot{\kappa}$) for the cyclic bending test were 0.0035, 0.035 and 0.35 $m^{-1}s^{-1}$. Two load cells installed in the testing facility (Fig. 3) were used to measure the bending moment. The light-weight instrument in Fig. 3 was used to measure the curvature and ovalization. The number of bending cycles required to ignite buckling was also recorded.

3. Experimental and Theoretical Results

The experimental data of cyclic curvature (κ_c) versus the number of bending cycles required to ignite buckling (N_b), for local sharp-notched SUS304 stainless steel tubes submitted to cyclic bending with $a = 0.2, 0.4, 0.6, 0.8,$ and 1.0 mm at $\dot{\kappa} = 0.0035, 0.035$ and $0.35 \text{ m}^{-1}\text{s}^{-1}$, respectively, is illustrated in Figs. 5-7 [1]. For a fixed κ_c and a , a larger $\dot{\kappa}$ causes a lower N_b . Next, Figs. 5-7 show the same data of Figs. 8-10 plotted on the log-log scale in dot lines. Note that the dot lines were least square fits of the data. It can be seen that for a fixed $\dot{\kappa}$, five almost parallel dot lines corresponding to five different a . In addition, the slopes of the parallel dot lines for three different $\dot{\kappa}$ are almost the same.

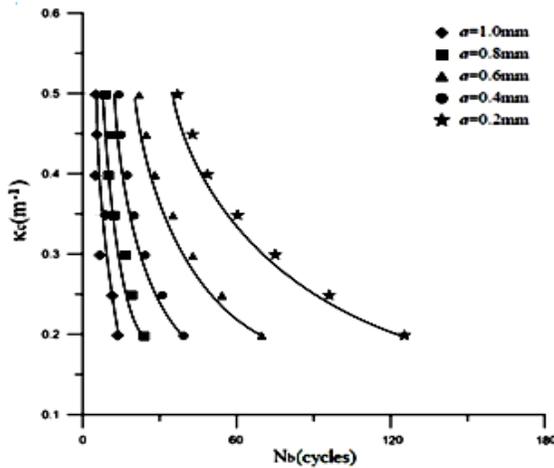


Fig. 5 Experimental controlled curvature (κ_c) - number of bending cycles required to ignite buckling (N_b) curves for local sharp-notched SUS304 stainless steel tubes with $a = 0.2, 0.4, 0.6, 0.8,$ and 1.0 mm under cyclic bending at $\dot{\kappa} = 0.0035 \text{ m}^{-1}\text{s}^{-1}$.

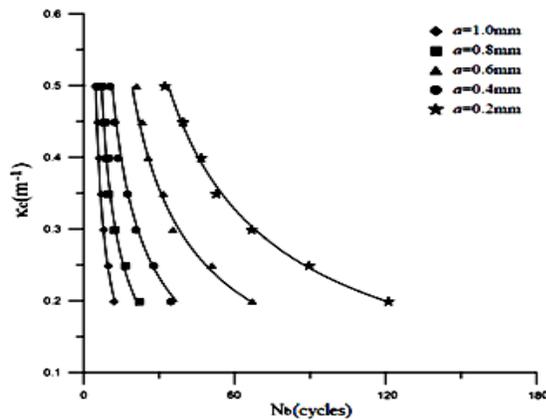


Fig. 6 Experimental controlled curvature (κ_c) - number of bending cycles required to ignite buckling (N_b) curves for local sharp-notched SUS304 stainless steel tubes with $a = 0.2, 0.4, 0.6, 0.8,$ and 1.0 mm under cyclic bending at $\dot{\kappa} = 0.035 \text{ m}^{-1}\text{s}^{-1}$.

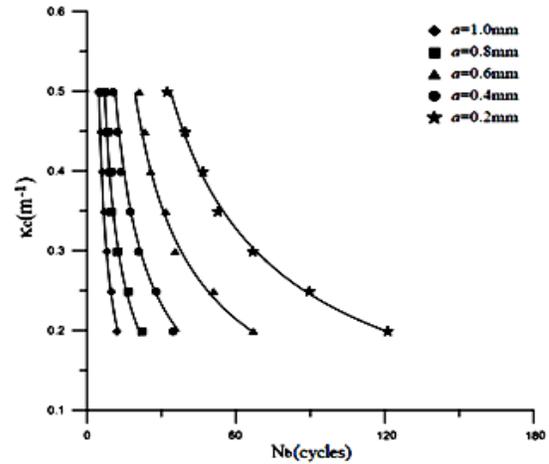


Fig. 7 Experimental controlled curvature (κ_c) - number of bending cycles required to ignite buckling (N_b) curves for local sharp-notched SUS304 stainless steel tubes with $a = 0.2, 0.4, 0.6, 0.8,$ and 1.0 mm under cyclic bending at $\dot{\kappa} = 0.35 \text{ m}^{-1}\text{s}^{-1}$.

In 1987, Kyriakides and Shaw [2] proposed an empirical form of the relationship between κ_c and N_b to be

$$\kappa_c = C (N_b)^{-\alpha} \quad (1)$$

or

$$\log \kappa_c = \log C - \alpha \log N_b \quad (2)$$

where C and α are the material parameters. The quantity C is the value of κ_c by letting $N_b = 1$, and the quantity α is the slope of the line in the log-log plot. In our study, due to same slope for every a at different $\dot{\kappa}$ in all $\log \kappa_c$ - $\log N_b$ relationships, the magnitude of α was determined to be 0.69.

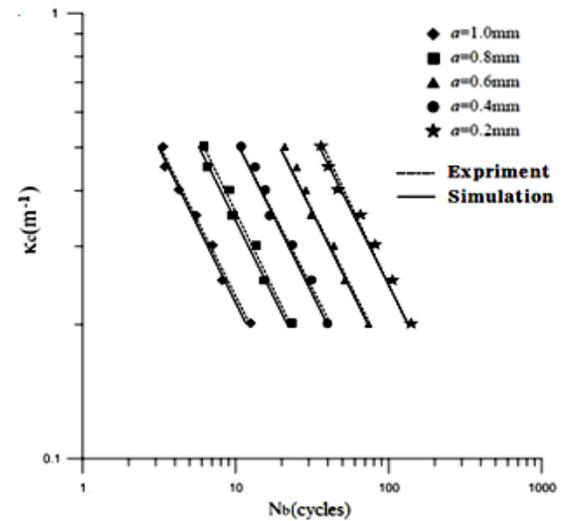


Fig. 8 Experimental and simulated controlled curvature (κ_c) - number of bending cycles required to ignite buckling (N_b) curves for local sharp-notched SUS304 stainless steel tubes with $a = 0.2, 0.4, 0.6, 0.8,$ and 1.0 mm under cyclic bending at $\dot{\kappa} = 0.0035 \text{ m}^{-1}\text{s}^{-1}$ on a log-log scale.

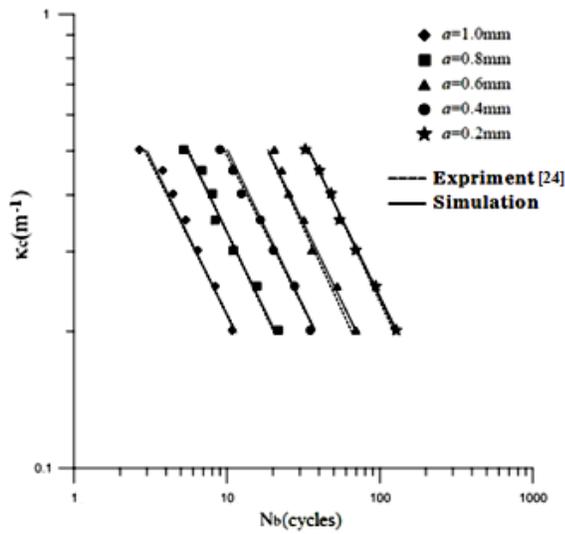


Fig. 9 Experimental and simulated controlled curvature (κ_c) - number of bending cycles required to ignite buckling (N_b) curves for local sharp-notched SUS304 stainless steel tubes with $a = 0.2, 0.4, 0.6, 0.8,$ and 1.0 mm under cyclic bending at $\dot{\kappa} = 0.035 \text{ m}^{-1}\text{s}^{-1}$ on a log-log scale.

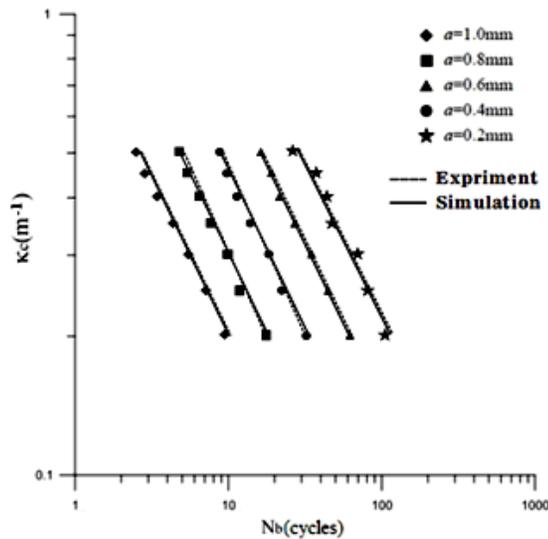


Fig. 10 Experimental and simulated controlled curvature (κ_c) - number of bending cycles required to ignite buckling (N_b) curves for local sharp-notched SUS304 stainless steel tubes with $a = 0.2, 0.4, 0.6, 0.8,$ and 1.0 mm under cyclic bending at $\dot{\kappa} = 0.35 \text{ m}^{-1}\text{s}^{-1}$ on a log-log scale.

Based on the experimental data in Figs. 8-10, five values of C were obtained for $a = 0.2, 0.4, 0.6, 0.8,$ and 1.0 mm at $\dot{\kappa} = 0.0035, 0.035,$ and $0.35 \text{ m}^{-1}\text{s}^{-1}$ from Eq. (2), as shown in Table 1. For the curvature rate effect of smooth SUS304 stainless steel tubes, an empirical form of C was proposed by Pan and Her [3] to be

$$C = C_o + C_1 \left[\log \frac{\dot{\kappa}}{\dot{\kappa}_o} \right]^2, \quad (3)$$

where C_o and C_1 are the material parameters, $\dot{\kappa}_o$ is the lowest curvature rate and $\dot{\kappa}$ is the other curvature rate. Lee et al. [11] has investigated the effect of a on κ_c - N_b relationship and proposed the formulation of C to be

$$\log C = c_1 \left(\frac{a}{t} \right) + c_2 \quad (4)$$

or

$$C = 10^{c_1 \left(\frac{a}{t} \right) + c_2}, \quad (5)$$

where c_1 and c_2 are material parameters.

For considering both a and $\dot{\kappa}$ effects, a simple linear combination of Eqs. (3) and (5) was proposed to be

$$C = 10^{c_1 \left(\frac{a}{t} \right) + c_2} + c_3 \left[\log \frac{\dot{\kappa}}{\dot{\kappa}_o} \right]^2, \quad (6)$$

where c_3 is material parameter. For smooth tubes, the magnitude of a is equal to 0. Thus, the value of 10^{c_2} is equal to C_o and C_1 is equal to c_3 , Eq. (6) is equal to Eq. (3). Next, by letting $\dot{\kappa} = \dot{\kappa}_o$, the second term of Eq. (6) is equal to zero. Thus, Eq. (6) is the same as Eq. (5).

Table 1: Experimental magnitudes of C for every a at different $\dot{\kappa}$

	$a = 0.2$ mm	$a = 0.4$ mm	$a = 0.6$ mm	$a = 0.8$ mm	$a = 1.0$ mm
$\dot{\kappa} = 0.0035$ $\text{m}^{-1}\text{s}^{-1}$	5.75	4.08	2.51	1.81	1.17
$\dot{\kappa} = 0.035$ $\text{m}^{-1}\text{s}^{-1}$	5.72	4.04	2.74	1.79	1.14
$\dot{\kappa} = 0.35$ $\text{m}^{-1}\text{s}^{-1}$	5.25	3.54	2.29	1.51	1.01

In our study, parameters c_1, c_2 and c_3 in Eq. (6) were determined by the following steps. First step, by letting $\dot{\kappa} = 0.0035 \text{ m}^{-1}\text{s}^{-1}$ which is equal to the lowest curvature rate $\dot{\kappa}_o$, Eq. (6) is then equal to Eq. (4). From the experimental data for the relationship between $\log C$ and a/t in Fig. 11, the straight line is obtained by the least square method. Thus, parameters c_1 and c_2 in Eq. (4) can be obtained to be -1.13 and 0.91, respectively. Second step, according to the variation of $\log C$ and a/t for $\dot{\kappa} = 0.035$ and $0.35 \text{ m}^{-1}\text{s}^{-1}$ in Fig. 11, the value of c_3 is adjusted to fit the experimental data. Thus, the parameter of c_3 can be obtained to be -0.11. Figs. 8-10 depict the simulated controlled curvature (κ_c) - number of bending cycles required to ignite buckling (N_b) curves for local sharp-notched SUS304 stainless steel tubes with $a = 0.2, 0.4, 0.6, 0.8,$ and 1.0 mm submitted to cyclic bending at $\dot{\kappa} = 0.0035, 0.035$ and $0.35 \text{ m}^{-1}\text{s}^{-1}$, respectively, in solid lines. It can be seen that good agreement between the experimental and simulated results has been achieved.

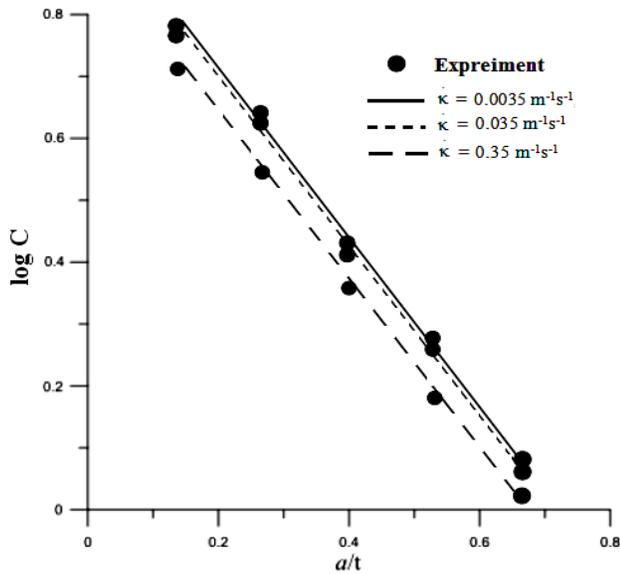


Fig. 11 Relationship between $\log C$ and a/t for $\dot{\kappa} = 0.0035, 0.035$ and $0.35 \text{ m}^{-1}\text{s}^{-1}$.

4. Conclusion

The theoretical simulation of the collapse of local sharp-notched SUS304 stainless steel tubes with different a submitted to cyclic bending at different $\dot{\kappa}$ are discussed in this study. The formula, Eq. (1), suggested by Kyriakides and Shaw [2] was modified to simulate the correlation between κ_c and N_b for local sharp-notched SUS304 stainless steel tubes with different a submitted to cyclic bending at different $\dot{\kappa}$. Due to same slope for every a at different $\dot{\kappa}$ in all $\log \kappa_c$ - $\log N_b$ relationships, the magnitude of α was determined to be 0.69. In this study, for considering both a and $\dot{\kappa}$ effects, a simple linear combination of the parameter C was proposed in Eq. (6). It can be observed that the simulations by Eqs. (1) and (6) are in good correlation with the experimental results [1] as shown in Figs. 8-10.

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